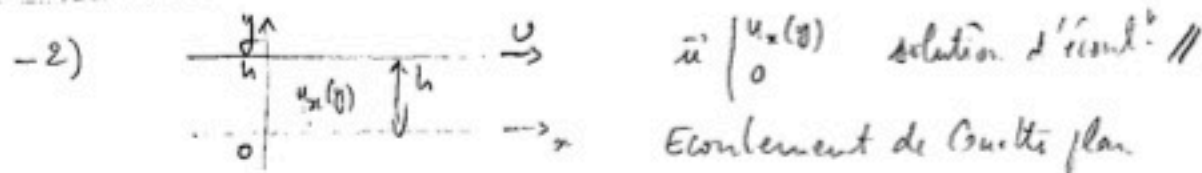


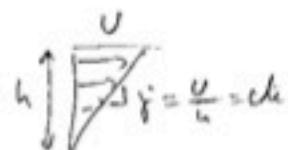
Plaque de mer

Partie I : Fluide visqueux

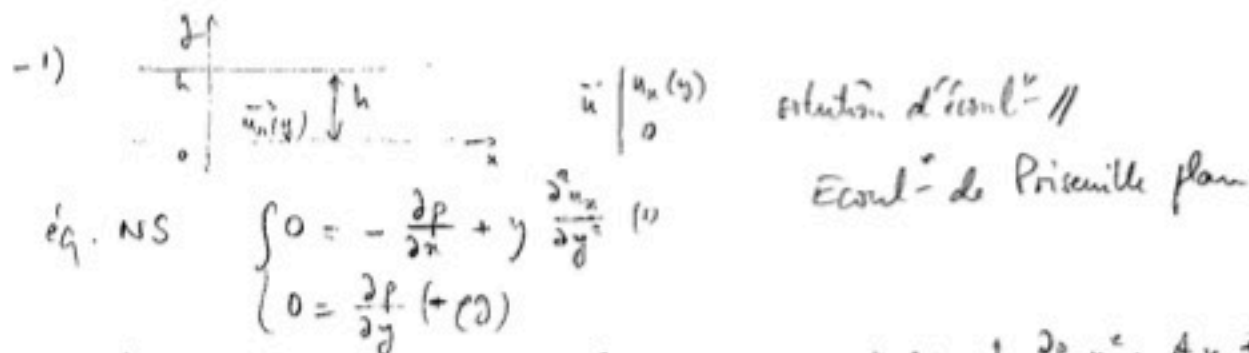


Éq. NS $\begin{cases} 0 = \eta \frac{\partial^2 u_x}{\partial y^2} \quad (1) & \text{car } \frac{\partial p}{\partial x} = 0 \\ 0 = \frac{\partial p}{\partial y} \quad (2) \end{cases}$

(1) $\Rightarrow \frac{\partial^2 u_x}{\partial y^2} = 0 \Rightarrow \frac{\partial u_x}{\partial y} = A \quad u_x(y) = Ay + B.$


ZCL: $\begin{cases} u_x(y=0) = 0 \Rightarrow B = 0 \\ u_x(y=h) = U \Rightarrow A = \frac{U}{h} \end{cases} \Rightarrow u_x(y) = \frac{U}{h} y$ 

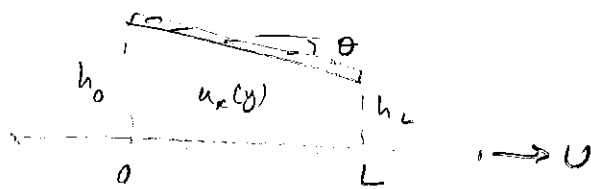
$\sigma_{xy} = \eta \frac{\partial u_x}{\partial y} = \eta A = \eta \frac{U}{h}$ indépendant de y



Éq. NS $\begin{cases} 0 = -\frac{\partial p}{\partial x} + \eta \frac{\partial^2 u_x}{\partial y^2} \quad (1) \\ 0 = \frac{\partial p}{\partial y} \quad (2) \end{cases}$

(1) $\Rightarrow \frac{\partial^2 u_x}{\partial y^2} = \frac{1}{\eta} \frac{\partial p}{\partial x} \Rightarrow \frac{\partial u_x}{\partial y} = \frac{1}{\eta} \frac{\partial p}{\partial x} y + A \Rightarrow u_x(y) = \frac{1}{2\eta} \frac{\partial p}{\partial x} y^2 + Ay + B$

ZCL $\begin{cases} u_x(y=0) = 0 \Rightarrow B = 0 \\ u_x(y=h) = 0 \Rightarrow A = -\frac{1}{2\eta} \frac{\partial p}{\partial x} h \end{cases} \Rightarrow u_x(y) = -\frac{1}{2\eta} \frac{\partial p}{\partial x} y(h-y)$ 



1) $h(x) = h_0 - \theta x$ avec $\theta = \frac{h_0 - h_L}{L}$ car $\tan \theta \approx \theta$ si $\theta \ll 1$

2) $u_x \sim U$ et $u_y \approx \theta u_x \sim \theta U$

3) Si $Re \ll \frac{1}{\theta}$ alors termes inertiels négligeables devant terme visqueux prépondérant

Eq. NS simplifiés $\begin{cases} 0 = -\frac{\partial p}{\partial x} + \eta \frac{\partial^2 u_x}{\partial y^2} & (1) \\ 0 = \frac{\partial p}{\partial y} & (2) \end{cases}$

(2) \Rightarrow p indépendant de y car $u_y \approx 0$ car $u_y \ll u_x$.

$\frac{\partial p}{\partial x}$ car u_x dépend de x car non //.

(1) $\Rightarrow \frac{\partial^2 u_x}{\partial y^2} = \frac{1}{\eta} \frac{\partial p}{\partial x} \Rightarrow u_x(y) = \frac{1}{2\eta} \frac{\partial p}{\partial x} y^2 + Ay + B$

z.c.l. $u_x(y=0) = U \Rightarrow B = U$

$u_x(y=h) = 0 \Rightarrow A = -\frac{1}{2\eta} \frac{\partial p}{\partial x} h - \frac{U}{h}$

$u_x(y) = \underbrace{-\frac{1}{2\eta} \frac{\partial p}{\partial x} y(h-y)}_{\text{Poiseuille}} + \underbrace{\frac{U}{h}(h-y)}_{\text{Couette}}$

Superposition car eq. NS linéaire sans les termes non-linéaires.

4) $Q = \int_0^h u_x(y) \ell dy = \ell \left[-\frac{1}{2\eta} \frac{\partial p}{\partial x} \left(\frac{hy^2}{2} - \frac{y^3}{3} \right) + \frac{U}{h} \left(hy - \frac{y^2}{2} \right) \right]_0^h$
 $= \ell \left[-\frac{1}{2\eta} \frac{\partial p}{\partial x} \left(\frac{h^3}{2} - \frac{h^3}{3} \right) + \frac{U}{h} \left(h^2 - \frac{h^2}{2} \right) \right] = \ell \left[\frac{Uh}{2} - \frac{h^3}{12\eta} \frac{\partial p}{\partial x} \right]$

Q ne dépend pas de x car écoulement incompressible stationnaire.

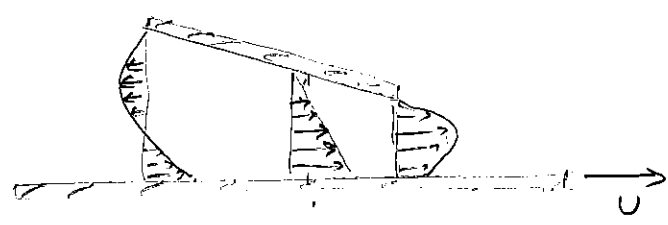
5) $\frac{\partial p}{\partial x} = \frac{12\eta}{h^3} \left[\frac{Uh}{2} - \frac{Q}{\ell} \right]$ or $\frac{\partial p}{\partial h} = \frac{\partial p}{\partial x} \frac{\partial x}{\partial h}$ avec $\frac{\partial x}{\partial h} = -\frac{1}{\theta}$

$\Rightarrow \frac{\partial p}{\partial h} = -\frac{1}{\theta} \frac{\partial p}{\partial x} = \frac{12\eta}{\theta h^3} \left[\frac{Q}{\ell} - \frac{Uh}{2} \right]$

6) z.c.l. $\begin{cases} p = p_{at} \text{ en } h = h_0 \\ p = p_{at} \text{ en } h = h_L \end{cases} \Rightarrow Q =$

$P(h) = p_{at} + \frac{6\eta U}{\theta} \frac{(h_0 - h)(h - h_L)}{h^2(h_0 + h_L)}$

7)



$u_x(y)$ ne dépend pas de la viscosité car régime stationnaire \Rightarrow couches limites stabilisées

8)

$$F_N = \int_0^x (p - p_{at}) l dx = \frac{-1}{\theta} \int_{h_L}^{h_0} \frac{6\eta U l}{\theta} \frac{(h_0 - h)(h - h_L)}{h^2 (h_0 + h_L)} dh$$

$$F_N \sim \frac{6\eta U l}{\theta^2} \int_{h_L}^{h_0} \frac{(h_0 - h)(h - h_L)}{h^2 (h_0 + h_L)} dh \quad \text{avec } \int = \frac{1}{h_0 + h_L} \left[\frac{h_0 h_L}{h} \right]$$

9)

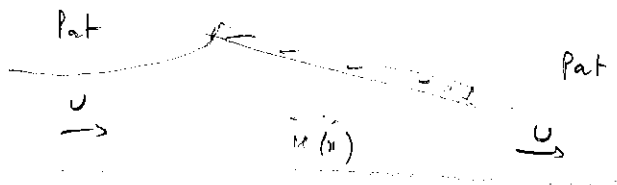
$$F_T = \int_0^L \eta \left. \frac{\partial u_x}{\partial y} \right|_{y=0} l dx \quad \text{avec} \quad \left. \frac{\partial u_x}{\partial y} \right|_{y=0} = -\frac{1}{2\eta} \frac{\partial p}{\partial x} h = \frac{h}{2\eta\theta} \frac{\partial p}{\partial x} = \frac{U}{h} =$$

$$F_T \sim \frac{6\eta U l}{\theta} \quad \frac{F_N}{F_T} \sim \frac{1}{\theta} \gg 1 \quad \Rightarrow \quad F_N \gg F_T$$

10)

$$Re = \frac{\rho U h}{\eta}$$

Partie II : Fluide parfait



1) Fluide parfait : viscosité nulle

2) Glissement total pour un fluide parfait à la paroi \Rightarrow pas d'entraîné possible \Rightarrow la vitesse éventuelle du sol ne change rien au problème

3) Cons. du débit : $u(x)h(x) = Uh_L \Rightarrow u(x) = U \frac{h_L}{h(x)}$

4) Bernoulli (cons. de l'énergie) : $\frac{1}{2} \rho u(x)^2 + p(x) = \frac{1}{2} \rho U^2 + p_{at}$ sur une ligne de courant à $z = \text{cte}$ entre x et L

$$p(x) = p_{at} + \frac{1}{2} \rho (U^2 - u(x)^2) = p_{at} + \frac{1}{2} \rho U^2 \left(1 - \left(\frac{u(x)}{U} \right)^2 \right)$$

$$p(x) = p_{at} + \frac{1}{2} \rho U^2 \left[1 - \frac{h_L^2}{h(x)^2} \right]$$

5° $F_p = \cos \theta \int_0^L [p(x) - p_{at}] dx = -\frac{\rho}{\theta} \int_{h_L}^{h_0} \frac{1}{2} \rho U^2 \left(1 - \frac{h_L^2}{h^2} \right) dh$

$$F_p = \frac{\rho U^2 l}{2\theta} \int_{h_L}^{h_0} \left(1 - \frac{h_L^2}{h^2} \right) dh = \frac{\rho U^2 l}{2\theta} \left[h + \frac{h_L^2}{h} \right]_{h_L}^{h_0}$$

$$= \frac{\rho U^2 l}{2\theta} \left[h_0 - h_L + \frac{h_L^2}{h_0} - h_L \right]$$

ou $h_0 - h_L = \theta L$

$$= \frac{\rho U^2 l}{2\theta} (h_0 - h_L) \left[1 - \frac{h_L}{h_0} \right]$$

$$F_p = \frac{\rho U^2 L l}{2} \left(1 - \frac{h_L}{h_0} \right)$$

$$F_p = C_p \frac{1}{2} \rho U^2 L l \text{ avec } C_p = 1 - \frac{h_L}{h_0} = \frac{\theta L}{h_0}$$

6° $F_F = \sin \theta \int_0^L [p(x) - p_{at}] dx = \theta F_p$

$$\frac{F_F}{F_N} = \frac{1}{\theta} \gg 1$$

7°

III Analyse dimensionnelle

1° $\rho, \nu, L, l, h_1, h_2 + F_0$

$[F] = \text{kg} \cdot \text{m} \cdot \text{s}^{-2}$

$[\rho] = \text{kg} \cdot \text{m}^{-3}$

$[\nu] = \text{m} \cdot \text{s}^{-1}$

$[L] = \text{m}$

$[h_i] = \text{m}$

$[h_2] = \text{m}$

$[l] = \text{m}$

7-3 = 4 paramètres sans dimensions

$\pi_1 = F \rho^\alpha \nu^\beta h_1^\gamma$

$\pi_1 = \frac{F}{\rho \nu^2 h_1^2}$

$\pi_2 = \frac{L}{h_1} \quad \pi_3 = \frac{h_2}{h_1} \quad \pi_4 = \frac{l}{h_1}$

$F = \rho \nu^2 h_1^2 f\left(\frac{L}{h_1}, \frac{l}{h_1}, \frac{h_2}{h_1}\right)$

si $F \propto L$ et $F \propto l$

$F = \rho \nu^2 L l f\left(\frac{h_2}{h_1}\right)$

$\frac{h_2}{h_1} = 0$

2° $\gamma, \rho, \nu, L, l, h_1, h_2 + F_p$

$\pi_1 = F \gamma^\alpha \nu^\beta h_1^\gamma$

$\pi_1 = \frac{F}{\gamma \nu h_1}$

$\pi_2 = \frac{L}{h_1} \quad \pi_3 = \frac{h_2}{h_1} \quad \pi_4 = \frac{l}{h_1}$

$F = \gamma \nu h_1 f\left(\frac{L}{h_1}, \frac{l}{h_1}, \frac{h_2}{h_1}\right)$

si $F \propto l$

$F = \gamma \nu l f\left(\frac{L}{h_1}, \frac{h_2}{h_1}\right)$

3° $\gamma, \rho, \nu, L, l, h_1, h_2 + F_p$

5 paramètres sans dimensions

$F = \underbrace{\gamma \nu h_1}_{\text{ou } \rho \nu^2 h_1^2} f\left(\frac{L}{h_1}, \frac{l}{h_1}, \frac{h_2}{h_1}, Re\right)$

avec $Re = \frac{\rho \nu h_1}{\gamma}$

$$L = 1 \text{ m} \quad l = 0,5 \text{ m} \quad U = 10 \text{ m/s} \quad h_1 = 1 \text{ cm} \quad h_2 = 2 \text{ cm}$$

$$\theta = \frac{10^{-2}}{1} = 10^{-2} \text{ rad}$$

$$\frac{F_p}{F_T} = 10^2$$

$$F_{p_v} = \frac{6\gamma U l}{\theta^2} = \frac{6 \times 10^{-3} \times 10 \times 0,5}{10^{-4}} = 300 \text{ N} = 30 \text{ kgf}$$

$$\frac{F_{p_v}}{F_{p_i}} = \frac{1}{100}$$

$$F_{p_i} = \frac{1}{2} \rho U^2 L l = \frac{10^3 \times 10^2 \times 1 \times 0,5}{2} \approx 3 \cdot 10^4$$

$$Re = \frac{\rho U h}{\eta} = \frac{10^3 \times 10 \times 10^{-2}}{10^{-3}} = 10^5 \gg 1$$

$$\frac{F_{p_i}}{F_{p_v}} = \frac{\rho U L}{\eta \theta^2} = \frac{\rho U h}{\eta} \frac{L}{h} \frac{1}{\theta^2}$$

$$L = 1 \text{ m} \quad l = 0,5 \text{ m} \quad U = 1 \text{ m/s} \quad h_1 = 1 \text{ mm} \quad h_2 = 2 \text{ mm}$$

$$\theta = \frac{10^{-3}}{1} = 10^{-3} \text{ rad}$$

$$\frac{F_p}{F_T} = 10^3$$

$$F_{p_v} = \frac{6\gamma U l}{\theta^2} = \frac{6 \times 10^{-3} \times 1 \times 0,5}{10^{-6}} = 3 \cdot 10^3 = 300 \text{ kgf}$$

$$\frac{F_{p_v}}{F_{p_i}} = 10$$

$$F_{p_i} = \frac{1}{2} \rho U^2 L l = \frac{10^3 \times 1 \times 1 \times 0,5}{2} = 3 \cdot 10^2$$

$$Re = \frac{\rho U h}{\eta} = \frac{10^3 \times 1 \times 10^{-3}}{10^{-3}} = 10^3 \gg 1 \text{ mais}$$

$$Re = \frac{1}{\theta}$$

$$5. \frac{d}{dt} \iiint_{\Omega} \rho \vec{u} dt = \iiint_{\Omega} \frac{\partial(\rho \vec{u})}{\partial t} dt + \iint_S \rho \vec{u} (\vec{u} \cdot d\vec{s}) = \sum \vec{F}$$

$$\iint_S \rho \vec{u} (\vec{u} \cdot d\vec{s}) = -\rho U_A^2 h_A l + \rho U^2 h_1 l = \rho U$$

$$\sum \vec{F} = p_A h_A l - p_{at} h_1 l + F_T$$

$$F_T + p_A h_A l - p_{at} h_1 l = \rho U^2 h_1 l - \rho U_A^2 h_A l$$

$$\text{or per Bernoulli } \frac{1}{2} \rho U_A^2 + p_A = \frac{1}{2} \rho U^2 + p_{at}$$

$$F_T = l (\rho U^2 h_1 - \rho U_A^2 h_A - p_A h_A + p_{at} h_1)$$

$$\text{or } U h_1 = U_A h_A$$

$$U^2 h_1^2 = U_A^2 h_A^2$$

$$F_T = l \left(\rho U^2 h_1 - \rho U^2 h_1 \frac{h_1}{h_A} - p_A h_A + p_{at} h_1 \right) \quad h_A = h_1 + \theta L$$

$$= l \left[\rho U^2 h_1 \left(1 - \frac{h_1}{h_A} \right) - p_{at} h_A + \frac{1}{2} \rho U^2 \left(h_1 - \frac{h_1^2}{h_A} \right) + p_{at} h_1 \right]$$

$$= l \left[\rho U^2 h_1 \left(1 - \frac{h_1}{h_A} \right) - p_{at} \theta L - \frac{1}{2} \rho U^2 h_1 + \frac{1}{2} \rho U^2 \frac{h_1^2}{h_A} \right]$$

$$= l \left[\rho U^2 h_1 - \frac{1}{2} \rho U^2 h_1 - p_{at} \theta L \right]$$

$$= l \rho U^2 (h_1 - h_A)$$

$$\text{or } h_1 - h_A = \theta L$$

$$= \theta \rho U^2 L l$$